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**Sir Issac Newton College of Engineering and Technology-Nagapattinam**

**MODEL EXAM**

Fifth Semester

Mechanical Engineering

ME8593-DESIGN OF MACHINE ELEMENTS

Date: 01.12.2022  
Time: 9.30 -12.30 AM

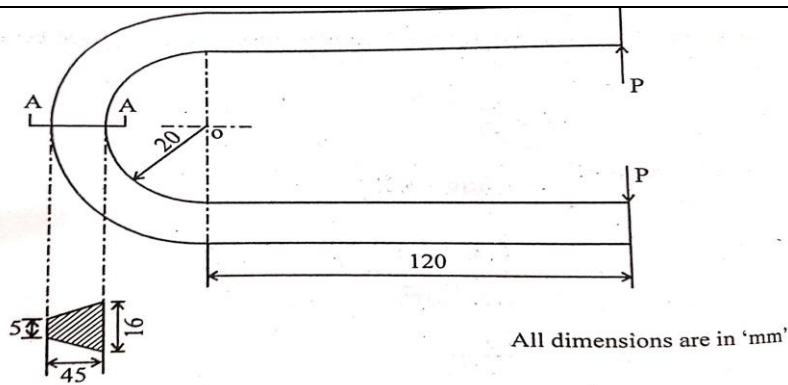
Session : FN  
Maximum: 100 marks

**Answer ALL Questions**  
**PART - A (10X 2 = 20 marks)**

Q.NO	Questions	Marks	CO	BL	PO
1	Define Endurance limit.	2	CO1	L3	3
2	What are the factors affecting endurance strength of a material	2	CO1	L2	3
3	How hollow shaft are superior to solid shaft?	2	CO2	L2	3
4	Is what situation flexible coupling are used?	2	CO2	L4	3
5	List the advantages and disadvantages of threaded fasteners.	2	CO3	L2	3
6	How is welding classified?	2	CO3	L3	3
7	Define Resilience of a spring.	2	CO4	L5	3
8	What is the function of a flywheel?	2	CO4	L2	3
9	State the merits of hydrostatic bearing	2	CO5	L3	3
10	What are the bearing materials?	2	CO5	L2	3

**PART- B (5 X 13 = 65 marks)**

Q.NO	Questions	Marks	CO	BL	PO
11.	(a). a frame of punch press is shown in figure. the frame is subjected to a load of 4.5kn as shown in figure. find the stress at the inner fiber and outer fiber of the frame.	13	CO1	L1	3



Or

(b). A cantilever beam of circular cross section is subjected to a cycle transverse load varying from  $-p$  to  $3p$  as shown in figure. the theoretical stress concentration factor is 1.4, notch sensitivity is 0.9, endurance limit is  $275 \text{ N/mm}^2$ , size factor is 0.85 and surface finish factor is 0.89 .using the above said values, find the maximum load that can be withstand by this cantilever beam. assume  $\sigma_u=550\text{N/mm}^2$ ,  $\sigma_y=470\text{N/mm}^2$  and factor of safety=2

13

CO1

L3

3

12. (a). A 15 kW, 960 r.p.m. motor has a mild steel shaft of 40 mm diameter and the extension being 75 mm. The permissible shear and crushing stresses for the mild steel key are 56 MPa and 112 MPa. Design the keyway in the motor shaft extension. Check the shear strength of the key against the normal strength of the shaft

13

CO2

L2

3

Or

(b). Design a clamp coupling to transmit 30 kW at 100 r.p.m. The allowable shear stress for the shaft and key is 40 MPa and the number of bolts connecting the two halves are six. The permissible tensile stress for the bolts is 70 MPa. The coefficient of friction between the muff and the shaft surface may be taken as 0.3

13

CO2

L3

3

13. (a) Design a knuckle joint to transmit 150 kN. The design stresses may be taken as 75 MPa in tension, 60 MPa in shear and 150 MPa in compression

13

CO3

L2

3

Or

(b). A plate 100 mm wide and 12.5 mm thick is to be welded to another plate by means of parallel fillet welds. The plates are subjected to a load of 50 kN. Find the length of the weld so that the maximum stress does not exceed 56 MPa. Consider the joint first under static loading and then under fatigue loading.

13

CO3

L3

3

14. (a). A helical spring is made from a wire of 6 mm diameter and has outside diameter of 75 mm. If the permissible shear stress is 350 MPa and modulus of rigidity  $84 \text{ kN/mm}^2$ , find the axial load which the spring can carry and the deflection per active turn.

13

CO4

L3

3

	<b>Or</b>				
	(b).. A shaft fitted with a flywheel rotates at 250 r.p.m. and drives a machine. The torque of machine varies in a cyclic inner over a period of 3 revolutions. The torque rises from 750 N-m to 3000 N-m uniformly during 1 / 2 revolution and remains constant for the following revolution. It then falls uniformly to 750 N-m during the next 1 / 2 revolution and remains constant for one revolution, the cycle being repeated thereafter. Determine the power required to drive the machine. If the total fluctuation of speed is not to exceed 3% of the mean speed, determine a suitable diameter and cross-section of the flywheel rim. The width of the rim is to be 4 times the thickness and the safe centrifugal stress is 6 MPa. The material density may be assumed as 7200 kg /m <sup>3</sup>	13	CO4	L2	3
15	(a). Design a journal bearing for a centrifugal pump from the following data: load on the journal 2000N; speed of the journal=900r.p.m; Type of oil is SAE 10, for which absolute viscosity at 55°C=0.017KG/M-S ambient temperature of oil =15.50°C; maximum bearing pressure of the pump=1.5 N/mm <sup>2</sup> . Calculate also mass of the lubricating oil required for artificial cooling, if rise of temperature of oil be limited to 10°C. heat dissipation coefficient=1232 W/m <sup>2</sup> /°C.	13	CO5	L4	3
	(b). A single row angular contact ball bearing number 310 is used for an axial flow compressor. The bearing is to carry a radial load of 2500 N and an axial or thrust load of 1500 N. Assuming light shock load, determine the rating life of the bearing.	13	CO5	L4	3

**PART- C (1 X 15 = 15 marks)**

Q.NO	Questions	Marks	CO	BL	PO
16.	(a). Design a sleeve and cotter joint to resist a tensile load of 60 kN. All parts of the joint are made of the same material with the following allowable stresses : $\sigma_t = 60 \text{ MPa}$ ; $\tau = 70 \text{ MPa}$ and $\sigma_c = 125 \text{ MPa}$	15	CO1	L3	3
	<b>Or</b>				
	(b) . Design a helical spring for a spring loaded safety valve (Rams bottom safety valve) for the following conditions. Diameter of valve seat = 65 mm ; Operating pressure = 0.7 N/mm <sup>2</sup> ; Maximum pressure when the valve blows off freely = 0.75 N/mm <sup>2</sup> ; Maximum lift of the valve when the pressure rises from 0.7 to 0.75 N/mm <sup>2</sup> = 3.5 mm: Maximum allowable stress = 550 MPa ;Modulus of rigidity = 84 kN/mm <sup>2</sup> ; Spring index = 6.	15	CO5	L4	3

BT LEVEL		CO1	CO2	CO3	CO4	CO5	CO6	%
<b>Remember</b>	<b>Q. Nos</b>	11(a),						7.22
	<b>Marks</b>	13						
<b>Understand</b>	<b>Q. Nos</b>	2	3,12(a)	5,13(a)	8,14(b)	10		27.22
	<b>Marks</b>	2	15	15	15	2		
<b>Applying</b>	<b>Q. Nos</b>	1,11(b), 16(a)	12(b)	6,13(b)	14(a)	9		40.55
	<b>Marks</b>	30	13	15	13	2		
<b>Analyze</b>	<b>Q. Nos</b>		4		15(a),15(b), 16(b)			23.88
	<b>Marks</b>		2		41			
<b>Evaluate</b>	<b>Q. Nos</b>				7			1.11
	<b>Marks</b>				2			
<b>Total</b>		45	30	30	71	4		100

<b>IQAC Member</b>	<b>HOD / Mech</b>